

CLAIMS

1. Rolling device (1) with at least two work rolls (2, 3), each of which is supported by a work roll chock (4, 5) in a rolling stand (6), wherein at least one of the work rolls (2, 3) in the rolling stand (6) can be adjusted, especially in the vertical direction, for the purpose of adjusting a desired roll gap relative to the other work roll (2, 3), wherein at least one work roll (2, 3) is operatively connected with bending devices (7), by which a bending moment can act on the work roll (2, 3), and wherein the work roll chock (4, 5) has arms (9, 10) that project laterally relative to the axis (8) of the work roll (2, 3) for absorbing the force produced by the bending devices (7), characterized by the fact that a pressure-transmitting element (12), which can be shifted relative to the rolling stand (6), especially in the vertical direction, is installed between an element (11) of the bending devices (7) that generates compressive force, especially a piston, and the projecting arm (9, 10) of the work roll chock (4, 5).

2. Rolling device in accordance with Claim 1, characterized by the fact that the element (11) of the bending devices (7) that generates compressive force and the projecting arm (9, 10) of the work roll chock (4, 5) are positioned in such

a way that the center axis (13) of the element (11) that generates compressive force intersects the projecting arm (9, 10).

3. Rolling device in accordance with Claim 1 or Claim 2, characterized by the fact that a sliding surface (14, 15) is arranged between the element (11) of the bending devices (7) that generates compressive force and the pressure-transmitting element (12) and/or between the pressure-transmitting element (12) and the projecting arm (9, 10) of the work roll chock (4, 5).

4. Rolling device in accordance with any of Claims 1 to 3, characterized by the fact that the bending devices (7) are mounted in a block (16) rigidly mounted on the rolling stand (6), and the pressure-transmitting element (12) is supported on the block (16) by means of a guide (17), especially a vertical guide.

5. Rolling device in accordance with Claim 4, characterized by the fact that the pressure-transmitting element (12) has a U-shaped horizontal cross section and surrounds the block (16), at least partially, on three sides.

6. Rolling device in accordance with Claim 4 or Claim 5, characterized by the fact that the pressure-transmitting element (12) has an L-shaped vertical cross section perpendicular to the

axis (8) of the work roll (2, 3) and at least partially surrounds the upper side of the block (16).

7. Rolling device in accordance with any of Claims 4 to 6, characterized by the fact that the pressure-transmitting element (12) is supported on the rolling stand (6) by means of a guide (18), especially a vertical guide.

8. Rolling device in accordance with any of Claims 4 to 7, characterized by the fact that holding devices (19) are installed between the block (16) and the pressure-transmitting element (12), which hold the pressure-transmitting element (12) stationary on the block (16) in the direction (R) towards the work roll (2, 3).

9. Rolling device in accordance with any of Claims 1 to 8, characterized by the fact that the work rolls (2, 3) are provided with axial shifting devices (20) for axial shifting of the work rolls (2, 3), with which the work rolls (2, 3) can be brought into a desired axial position relative to the rolling stand (6) and held there.

10. Rolling device in accordance with any of Claims 1 to 9, characterized by the fact that the extent of the projecting arm (9, 10) of the work roll chock (4, 5) in the direction of the axis (8) of the work roll (2, 3) is large in relation to the extent of the pressure-transmitting element (12) measured in the

direction of the axis (8) at its part that is connected with the projecting arm (9, 10), preferably at least twice as large.

11. Rolling device in accordance with any of Claims 1 to 9, characterized by the fact that the extent of the projecting arm (9, 10) of the work roll chock (4, 5) in the direction of the axis (8) of the work roll (2, 3) is small in relation to the extent of the pressure-transmitting element (12) measured in the direction of the axis (8) at its part that is connected with the projecting arm (9, 10) and preferably is no more than half as large.